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Design of a Two-Phase Loop Thermosyphon for Telecommunications System (]]) - Analysis and Simulation -

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A computer simulation is performed for a two-phase loop thermosyphon for the B-ISDN telecommunications. The aim of this code development is to provide capabilities to predict the affects of many variables on the performance of the proposed TLT system using different empirical correlations obtained from the literature for the evaporation and condensation, and the shape factors available. In the present study, the simulation code is based on the sectorial thermal resistance network built on the flow regimes of the two-phase flows involved. The nodal resistances are solved by the typical Gauss-Seidal iteration method. The code can predict whether the proposed design is possible based on the flooding limit calculation of the system and its results are compared with the experimental results.

Key Words : Thermosyphon, Sectorial Thermal Resistance, Simulation, Two-Phase Flow, Shape Factor

| Nomenclature | | G | : Mass flux (kg/m ² s) |
|--|--|--------------|---|
| Α | : Area (m ²) | g | : Acceleration due to gravity (m ² /s) |
| Ar | : Flow area occupied by liquid phase | Н | : Specific enthalpy (J/kg) |
| | (m ²) | h | : Heat transfer coefficient (W/m ² K), |
| Ar | : Flow area occupied by vapour | | evaporator depth (m) |
| 0 | phase (m ²) | Hfluid | : Specific enthalpy of fluid (J/kg) |
| a | : Evaporator thickness (m) | H_f | : Specific enthalpy of saturated liquid |
| $A_{\mathbf{x}}$ | : Cross sectional area (m ²) | | (J/kg) |
| b | : Evaporator width (m) | H_{fg} | : Latent heat of evaporation (J/kg) |
| С | Parameter | k | : Thermal conductivity (W/m K) |
| C_{Pl} | : Specific heat of saturated liquid | L | : Length (m) |
| | (J/kg K) | Nu | : Nusselt number |
| C_{SF} | : A constant on Rohsenow's pool | ΔP_a | : Acceleration pressure drop |
| | boiling correlation | ΔP_f | : Frictional pressure drop |
| D | : Outer diameter of tube (m), param- | ΔP_h | : Hydraulic pressure drop |
| | eter | Р | : Pressure (Pa) |
| d | : linner diameter (m) | (dP/dy) | p_{p} ; Pressure gradient |
| D_h | : Hydraulic diameter (m) | Pr | : Prandtl number |
| F | : Frictional parameter | Q | : Heat transfer rate (W) |
| f | : Friction factor | q | : Heat flux (W/cm ²) |
| * Department of Mechanical Engineering University of | | R | : Resistance (K/W) |
| Ottawa, Ottawa, Ontario, Canada | | r | : Radius (m) |
| ** Department of Industrial Engineering, Kongju | | Ra | : Rayleigh number |
| National University, Chungnam, Korea | | Re | : Reynolds number |
| Packaging rechnology Section, Electronics and Tele- | | S | : Shape factor, difined as eqn. (2) |

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